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There are two Problems and they are assigned 20 points each (for a total of 40/40)

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**Part #1 (20 points total):**

Consider an inertial system modeled as a lumped scalar linear system

$$\frac{d^2x(t)}{dt^2} = u(t).$$

Here  $u(t)$  represents a force applied to it by a controller, while  $x(t)$  represents its position at time  $t \in [0, \infty)$ . The controller input is taken as the difference  $v(t) = r(t) - x(t)$ , between a reference signal  $r(t)$  and the position  $x(t)$ . The controller output is the applied force  $u(t)$ .

**Address the following three independent sub-parts to Part 1:**

1. **(5 point)** Consider the control law  $u(t) = Kv(t)$  and determine whether the feedback system is asymptotically stable for any choice of the gain  $K$ . (Recall that a system is asymptotically stable if all of its poles have negative real part.)
2. **(5 point)** Consider a dynamic control law where

$$u(t) = v(t) - \int_0^t v(\tau)e^{-2(t-\tau)}d\tau.$$

Determine whether the feedback system is asymptotically stable for this choice of control law. (Prove that it is, or prove that it is not.)

3. **(10 points)** Design a control law which stabilizes the feedback system and has the additional property that, at steady state, the position follows exactly a sinusoidal reference signal  $r(t) = \sin(t)$ .

**Solution:**

1. The transfer function from  $u(t)$  to  $x(t)$  is  $P(s) = \frac{1}{s^2}$ . Hence the characteristic equation for the closed loop system with unity negative feedback and a gain  $K$  is:

$$1 + \frac{K}{s^2} = 0.$$

This equation has roots on the imaginary axis for all values of  $K \geq 0$ , and has at least one root in the right half plane for  $K < 0$ . Hence, the system is not asymptotically stable for any choice of  $K$ .

2. The control law can also be written as:

$$u(t) = v(t) - v(t) \star e^{-2t},$$

where  $\star$  denotes convolution. Evidently, its transfer function is

$$C(s) = 1 - \frac{1}{s+2}.$$

Hence, the characteristic equation of the closed loop system is

$$\begin{aligned} 0 &= 1 + \left(1 - \frac{1}{s+2}\right) \frac{1}{s^2} \\ &= 1 + \frac{s+1}{s+2} \frac{1}{s^2} \\ &= \frac{s^2(s+2) + (s+1)}{(s+2)s^2} \end{aligned}$$

Thus, the characteristic polynomial is

$$s^2(s+2) + (s+1) = s^3 + 2s^2 + s + 1.$$

The characteristic polynomial has all its roots in the left half plane as it can be readily verified using Routh's test:

$$\begin{array}{c|cc} s^3 & 1 & 1 \\ s^2 & 2 & 1 \\ s & 0.5 & \\ 1 & 1 & \end{array} \quad \text{with the first column having all positive entries.}$$

This proves that the feedback system is asymptotically stable.

3. The controller transfer function must have a pole at  $s = 1$ , so that the closed loop transfer function from  $r(t)$  to  $v(t)$ , namely,

$$S(s) = \frac{1}{1 + P(s)C(s)}$$

realizes a transmission zero at the frequency of the sinusoid. Thus,

$$C(s) = \frac{1}{s^2 + 1} \times \text{possibly more dynamics for stabilization.}$$

We notice that if we used a second order controller

$$C(s) = \frac{as^2 + bs + c}{s^2 + 1},$$

the characteristic polynomial for the feedback system would be

$$s^2(s^2 + 1) + as^2 + \dots = s^4 + (1+a)s^2 + \dots$$

with a zero coefficient for the term  $s^3$ . Therefore, the system would be unstable with such a controller. This suggests that we need a higher order controller. In this case there is a lot of flexibility on where for instance to place closed loop poles. Below is one particular solution.

We can choose a controller

$$C(s) = \frac{as^3 + bs^2 + cs + d}{(s^2 + 1)(s + f)}$$

in which case the characteristic polynomial is

$$s^5 + fs^4 + (1+a)s^3 + (f+b)s^2 + cs + d.$$

Clearly, we have enough authority to select the poles so that the characteristic polynomial is in fact equal to

$$(s+1)^5 = s^5 + 5s^4 + 10s^3 + 10s^2 + 5s + 1.$$

This choice requires that we take  $f = 5$ ,  $a = 9$ ,  $b = 5$ ,  $c = 5$ ,  $d = 1$ .

**Part #2 (20 points total):**

Consider a delay system, described by the delay-differential equation

$$\frac{d}{dt}x(t) = u(t) - Kx(t-1) - Kx(t-2),$$

where  $x(t)$  represents the state and  $u(t)$  the input. Determine the maximal value of  $K$  for which the system is stable. (Hint: You may use Nyquist's stability criterion.)

**Solution:**

The characteristic equation is

$$s + Ke^{-s} + Ke^{-2s} = 0.$$

Equivalently we may consider

$$1 + K\frac{e^{-s}}{s} + K\frac{e^{-2s}}{s} = 0$$

and we may consider the Nyquist diagram for loop gain with transfer function

$$K\frac{e^{-j\omega}}{j\omega} + K\frac{e^{-2j\omega}}{j\omega}.$$

Since for both terms in the transfer function of the loop gain the phase decreases with increasing  $\omega$ , we only need to find the phase crossover frequency (i.e., the frequency where the phase first becomes  $-\pi$  radians). Equivalently, we need to determine the frequency where the phase of

$$e^{-j\omega} + e^{-2j\omega}$$

is  $-\pi/2$  radians. At that frequency, the real part of  $e^{-j\omega} + e^{-2j\omega}$  must be zero, therefore

$$\cos(\omega) + \cos(2\omega) = 0.$$

The smallest value for which this is true is  $\omega = \frac{\pi}{3}$ . At this frequency, the absolute value of the loop gain is

$$K\frac{(\sin(\pi/3) + \sin(2\pi/3))}{\pi/3} = K\frac{\sqrt{3}}{\pi/3} = K\frac{3^{3/2}}{\pi}$$

and for stability of the system, the absolute value of the gain at the phase crossover frequency needs to be less than one. Hence, the maximal value for  $K$  is  $\frac{\pi}{3^{3/2}}$ .