EE 8235: Lecture 21

Lecture 21: Input-output analysis in fluid mechanics

• Linear analyses: Input-output vs. Stability



Transition in Newtonian fluids

- LINEAR HYDRODYNAMIC STABILITY: unstable normal modes
 - * **successful in:** Benard Convection, Taylor-Couette flow, etc.
 - * fails in: wall-bounded shear flows (channels, pipes, boundary layers)

DIFFICULTY #1 Inability to predict: Reynolds number for the onset of turbulence (Re_c)

Experimental onset of turbulence: $\begin{cases} much before instability \\ no sharp value for <math>Re_c \end{cases}$

DIFFICULTY #2 Inability to predict: flow structures observed at transition (except in carefully controlled experiments)

LINEAR STABILITY:

 $\begin{array}{l} \star \mbox{ For } Re \geq Re_c \ \Rightarrow \ \mbox{ exp. growing normal modes} \\ \mbox{ corresponding e-functions} \\ \mbox{ (TS-waves)} \end{array} \right\} \ := \ \mbox{ exp. growing flow structures} \end{array}$



EXPERIMENTS: streaky boundary layers and turbulent spots

 z_{\star}



Matsubara & Alfredsson, J. Fluid Mech. '01

• FAILURE OF LINEAR HYDRODYNAMIC STABILITY caused by high flow sensitivity

- ★ large transient responses
- ★ large noise amplification
- ★ small stability margins





Tools for quantifying sensitivity

• INPUT-OUTPUT ANALYSIS: spatio-temporal frequency responses





IMPLICATIONS FOR:

transition: insight into mechanisms

control: control-oriented modeling

Ensemble average energy density



 Dominance of streamwise elongated structures streamwise streaks!

Influence of *Re*: streamwise-constant model

$$\begin{bmatrix} \psi_{1t} \\ \psi_{2t} \end{bmatrix} = \begin{bmatrix} A_{os} & 0 \\ Re A_{cp} & A_{sq} \end{bmatrix} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix} + \begin{bmatrix} 0 & B_2 & B_3 \\ B_1 & 0 & 0 \end{bmatrix} \begin{bmatrix} d_1 \\ d_2 \\ d_3 \end{bmatrix}$$
$$\begin{bmatrix} u \\ v \\ w \end{bmatrix} = \begin{bmatrix} 0 & C_u \\ C_v & 0 \\ C_w & 0 \end{bmatrix} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix}$$
$$\overset{d_1}{=} \begin{bmatrix} 0 \\ C_w & 0 \end{bmatrix} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix}$$
$$\overset{d_1}{=} \begin{bmatrix} 0 \\ C_w & 0 \end{bmatrix} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix}$$
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$$\overset{d_1}{=} \begin{bmatrix} 0 \\ C_w & 0 \end{bmatrix} \begin{bmatrix} \psi_1 \\ \psi_2 \end{bmatrix}$$
$$\overset{d_1}{=} \begin{bmatrix} B_1 \\ (j\omega I - A_{os})^{-1} \\ (j\omega I - A_{os})^$$

Jovanović & Bamieh, J. Fluid Mech. '05

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Amplification mechanism in flows with high *Re*

• HIGHEST AMPLIFICATION: $(d_2, d_3) \rightarrow u$



AMPLIFICATION MECHANISM: vortex tilting or lift-up R



wall-normal direction



spanwise direction

Turbulence without inertia

NEWTONIAN: inertial turbulence

VISCOELASTIC: elastic turbulence



Groisman & Steinberg, Nature '00

NEWTONIAN:



VISCOELASTIC:



INFLOW RESISTANCE: increased 20 times!

Turbulence: good for mixing



Groisman & Steinberg, Nature '01

... bad for processing

POLYMER MELT EMERGING FROM A CAPILLARY TUBE



Kalika & Denn, J. Rheol. '87

CURVILINEAR FLOWS: purely elastic instabilities

Larson, Shaqfeh, Muller, J. Fluid Mech. '90

RECTILINEAR FLOWS: no modal instabilities





Oldroyd-B fluids

HOOKEAN SPRING:

$$(Re/We)\mathbf{v}_t = -Re(\mathbf{v}\cdot\nabla)\mathbf{v} - \nabla p + \beta \Delta \mathbf{v} + (1-\beta)\nabla\cdot\boldsymbol{\tau} + \mathbf{d}$$
$$0 = \nabla\cdot\mathbf{v}$$
$$\boldsymbol{\tau}_t = -\boldsymbol{\tau} + \nabla\mathbf{v} + (\nabla\mathbf{v})^T + We(\boldsymbol{\tau}\cdot\nabla\mathbf{v} + (\nabla\mathbf{v})^T\cdot\boldsymbol{\tau} - (\mathbf{v}\cdot\nabla)\boldsymbol{\tau})$$

VISCOSITY RATIO:

$$\beta := \frac{\text{solvent viscosity}}{\text{total viscosity}}$$

WEISSENBERG NUMBER: $We := \frac{\text{fluid relaxation time}}{\text{characteristic flow time}}$

REYNOLDS NUMBER:

$$Re := \frac{\text{inertial forces}}{\text{viscous forces}}$$

Input-output analysis



INSIGHT INTO AMPLIFICATION MECHANISMS
 importance of streamwise elongated structures

Hoda, Jovanović, Kumar, *J. Fluid Mech. '08, '09* Jovanović & Kumar, *JNNFM '11*

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Inertialess channel flow: worst case amplification

• No single constitutive equation can describe the range of phenomena

***** important to quantify influence of modeling imperfections on dynamics

$$G(k_{x},k_{z}) = \sup_{\omega} \sigma_{\max}^{2} \left(\mathcal{T}(k_{x},k_{z},\omega)\right):$$

$$We = 10, \beta = 0.5, Re = 0$$

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spanwise wavenumber

 $G(k_x, k_z)$:



 Dominance of streamwise elongated structures streamwise streaks!

Amplification mechanism

• Highest amplification: $(d_2, d_3) \rightarrow u$

INERTIALESS VISCOELASTIC:



Inertialess lift-up mechanism

$$\Delta \eta_t = -(1/\beta)\Delta \eta + We(1 - 1/\beta) A_{cp2} \vartheta$$
$$= -(1/\beta)\Delta \eta + We(1 - 1/\beta) \left(\partial_{yz}(U'(y)\tau_{22}) + \partial_{zz}(U'(y)\tau_{23})\right)$$



spanwise direction

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spanwise direction

Spatial frequency responses





Dominant flow patterns

• FREQUENCY RESPONSE PEAKS

streamwise vortices and streaks

Inertial Newtonian:

Inertialess viscoelastic:



• CHANNEL CROSS-SECTION VIEW:

color plots:streamwise velocitycontour lines:stream-function

Flow sensitivity vs. nonlinearity

• Challenge: relative roles of flow sensitivity and nonlinearity



• Newtonian fluids: self-sustaining process for transition to turbulence Waleffe, *Phys. Fluids '97*

